RESIDENTIAL ACM APPENDIX RD

Appendix R<u>D</u>K – Procedures for Determining Required Refrigerant Charge and Adequate Airflow for Split System Space Cooling Systems without Thermostatic Expansion Valves

RDK1 Purpose and ScopeOverview

Failure to maintain proper refrigerant charge or proper airflow across the coil reduces the seasonal energy efficiency for an air conditioner (whether a cooling only air conditioner or a heat pump). In addition, excessive refrigerant charge can cause premature compressor failure, while insufficient refrigerant charge allows compressors to overheat. Very low airflow can result in icing of the coil and compressor failure.

To help avoid these problems and to provide a compliance credit for correctly installed systems, The purpose of this this appendix describes procedures is to determine and verify that for determining if a residential split system space cooling systems and heat pumps have has the required refrigerant charge and adequate airflow across the evaporator coil. The applicability of these procedures have the following limitations: The procedures detailed in this appendix only apply to ducted split system central air conditioners and ducted split system central heat pumps that do not have thermostatic expansion valves (TXVs). As an alternative to the procedures detailed in this appendix, systems may substitute a TXV installed and confirmed through field verification and diagnostic testing. The procedures detailed in this appendix do not apply to single-packaged systems. For dwelling units with multiple split systems or heat pumps, the procedure shall be applied to each system separately.

Note that tThe procedures detailed in this appendix_ACM RD-2005 are intended to be used after the HVAC installer has installed and charged the <u>air conditioner or heat pump</u> system in accordance with the <u>manufacturer's specifications</u>. The installer shall install and charge the <u>air conditioner and heat pump equipment in accordance with the</u>-manufacturer's instructions and specifications for the specific model equipment installed. The installer shall certify to the builder, building official and HERS rater that <u>they have he/she has followed these the manufacturer's instructions</u> and specifications prior to proceeding with the procedures in this appendix.

AFor dwelling units with multiple systems, this procedure must be applied to each system separately.

This appendix ACM RD-2005 defines two procedures, the Standard Charge and Airflow Measurement procedure Procedure in Section RD2 and the Alternate Charge and Airflow Measurement Procedure in Section RD3. The Standard procedure shall be used when the outdoor air temperature is 55°F or above and shall always be used for HERS rater verification. HVAC installers who must complete system installation when the outdoor temperature is below 55°F shall use the Alternate procedure.

The following sections document the instrumentation needed, the required instrumentation calibration, the measurement procedure, and the calculations required for each procedure. Note: Wherever thermocouples appear in this document, thermisters can be used instead with the same requirements applying to thermisters as to thermocouples.

The reference method algorithms adjust (improve) the efficiency of split system air conditioners and heat pumps when they are diagnostically tested to have the correct refrigerant charge or when field verification indicates that a TXV has been installed. Table RD-1 summarizes the algorithms that are affected by refrigerant charge testing or field verification of a TXV.

<u>I able RD-</u> 1	<u>– Summary o</u>	<u>t Diagnostic Measureme</u>	<u>nts</u>		
	<u>Variables</u>			Proposed D	<u>esign</u>
Input to the	and Equation				
<u>Algorithms</u>	<u>Reference</u>	<u>Description</u>	Standard Design Value	<u>Default Value</u>	<u>P</u>
Cooling	_	F takes on a value of	Snlit systems are	No refrigerant charge	R

	<u>variabics</u>			1 10p0000 B	Joigi i
Input to the Algorithms	and Equation Reference	<u>Description</u>	Standard Design Value	<u>Default Value</u>	<u>Procedure</u>
Cooling System Refrigerant Charge	E _{TXV} (Eq. <u>R4-40</u> 4.42and <u>R4-</u> 414.43)	F _{TXV} takes on a value of 0.96 when the system has been diagnostically tested for the correct refrigerant charge. Otherwise, F _{TXV} has a value of 0.90.	Split systems are assumed to have refrigerant charge testing or a TXV, when required by Package D.	No refrigerant charge testing or TXV.	RD2 or RD3

Note that a prerequisite for diagnostically testing the refrigerant charge is to verify that there is adequate airflow over the evaporator coil. This diagnostic test is described in ACM RE-2005.

RDK2 Standard Charge and Airflow Measurement Procedure

This section specifies the Standard charge and airflow-measurement procedure. Under this procedure, required refrigerant charge is calculated using the Superheat Charging Method. and The method also checks adequate airflow across the evaporator coil is-to determine whether the charge test is valid calculated using the Temperature Split Method or the air flow measurement methods in ACM RE-2005.

The Standard procedure detailed in this section shall be completed when the outdoor temperature is 55°F or higher after the HVAC installer has installed and charged the system in accordance with the manufacturer's specifications. If the outdoor temperature is between 55°F and 65°F the return dry bulb temperature shall be maintained above 70°F during the test. All HERS rater verifications are required to use this Standard procedure.

RDK2.1 Minimum Qualifications for this Procedure

Persons carrying out this procedure need to shall be qualified to perform the following:

- Obtain accurate pressure/temperature readings from refrigeration manifold gauges.
- Obtain accurate temperature readings from thermometer and thermocouple set up.
- Check calibration of refrigerant gauges using a known reference pressure and thermometer/thermocouple set up using a known reference temperature.
- Determine best location for temperature measurements in ducting system and on refrigerant line set.
- Calculate the measured superheat and temperature split.
- Determine the correct level of superheat and temperature split required, based on the conditions present at the time of the test.
- Determine if measured values are reasonable.

RDK2.2 Instrumentation Specifications

Instrumentation for the procedures described in this section shall conform to the following specifications:

RDK2.2.1 Digital Thermometer

Digital thermometer must-shall have thermocouple compatibility (type K and J) and Celsius or Fahrenheit readout with:

- Accuracy: $\pm (0.1\% \text{ of reading} + 1.3^{\circ} \text{ F}).$
- Resolution: 0.2° F.

RDK2.2.2 Thermocouples

Measurements require five (5) heavy duty beaded low-mass wire thermocouples and one (1) cotton wick for measuring wet-bulb temperatures.

RDK2.2.3 Refrigerant Manifold Gauge Set

A standard multiport refrigerant manifold gauge with an accuracy of plus or minus 3% shall be used.

RDK2.3 Calibration

The accuracy of instrumentation shall be maintained using the following procedures. A sticker with the calibration check date shall be affixed to each instrument calibrated.

RDK2.3.1 Thermometer/Thermocouple Field Calibration Procedure

Thermometers/thermocouples shall be calibrated monthly to ensure that they are reading accurate temperatures. The following procedure shall be used to check thermometer/thermocouple calibration:

- 1. Step 1-Fill an insulated cup (foam) with crushed ice. The ice shall completely fill the cup. Add water to fill the cup.
- 2. Step 2 Insert two thermocouples into the center of the ice bath and attach them to the digital thermometer.
- 3. Step 3-Let the temperatures stabilize. The temperatures shall be 32°F (+/- 1°F). If the temperature is off by more than 1°F make corrections according to the manufacturer's instructions. Any thermocouples that are off by more than 3°F shall be replaced.
- 4. Step 4-Switch the thermocouples and ensure that the temperatures read on T1 and T2 are still within +/- 1°F of 32°F.
- 5. Step 5-Affix sticker with calibration check date onto thermocouple.
- 6. Step 6-Repeat the process for all thermocouples.

RDK2.3.2 Refrigerant Gauge Field Check Procedure

Refrigerant gauges shall be checked monthly to ensure that the gauges are reading the correct pressures and corresponding temperatures. The following procedure shall be used to check gauge calibration:

- 1. Step 1-Place a refrigerant cylinder in a stable environment and let it sit for 4 hours minimum to stabilize to the ambient conditions.
- 2. Step 2-Attach a thermocouple to the refrigerant cylinder using duct tape so that there is good contact between the cylinder and the thermocouple.
- 3. Step 3-Insulate the thermocouple connection to the cylinder (closed cell pipe insulation can be taped over the end of the thermocouple to provide the insulation).
- 4. Step 4-Zero the low side compound gauge with all ports open to atmospheric pressure (no hoses attached).
- 5. Step 5-Re-install the hose and attach the low side gauge to the refrigerant cylinder.
- 6. Step 6-Read the temperature of the thermocouple.
- 7. Step 7Using a pressure/temperature chart for the refrigerant, look up the pressure that corresponds to the temperature measured.
- 8. Step 8lf gauge does not read the correct pressure corresponding to the temperature, the gauge is out of calibration and needs to be replaced or returned to the manufacturer for calibration.
- 9. Step 9Repeat the process in steps 4 through 8 for the high side gauge.
- 10. Step 10Affix sticker with calibration check date onto refrigerant gauge.

RDK2.4 Charge and Airflow Measurements

The following procedure shall be used to obtain measurements necessary to adjust required refrigerant charge and adequate airflow as described in the following sections:

- Step 1.1. If the condensor air entering temperature is less than 65°F, Eestablish a return air dry bulb temperature sufficiently high that the return air dry bulb temperature will be not less than 70°F prior to the measurements at the end of the 15 minute period in step 2.
- 2. Step 2-Turn the cooling system on and let it run for 15 minutes to stabilize temperatures and pressures before taking any measurements. While the system is stabilizing, proceed with setting up the temperature measurements.
- 3. Step 3-Connect the refrigerant gauge manifold to the suction line service valve.
- 4. Step 4 Attach a thermocouple to the suction line near the suction line service valve. Be sure the sensor is in direct contact with the line and is well insulated from air temperature.
- 5. Step 5Attach a thermocouple to measure the condenser (entering) air dry-bulb temperature. The sensor shall be placed so that it records the average condenser air entering temperature and is shaded from direct sun.
- 6. Step 6-Be sure that all cabinet panels that affect airflow are in place before making measurements. The thermocouple sensors shall remain attached to the system until the final charge is determined.
- 7. Step 7Place wet-bulb thermocouple in water to ensure it is saturated when needed. **Do not get the dry-bulb thermocouples wet.**
- 8. Step 8Insert the dry-bulb thermocouple in the supply plenum at the center of the airflow.
- 9. Step 9At 12 minutes, insert a dry-bulb thermocouple and a wet-bulb thermocouple into the return plenum at the center of the airflow.
- 10. Step 10At 15 minutes when the return plenum temperatures have stabilized, using the thermocouples already in place, measure and record the return (evaporator entering) air dry-bulb temperature (T_{return, db}) and the return (evaporator entering) air wet-bulb temperature (T_{return, wb}).
- 11. Step 11-Using the dry-bulb thermocouple already in place, measure and record the supply (evaporator leaving) air dry-bulb temperature (T_{supply, db}).
- 12. Step 12 Using the refrigerant gauge already attached, measure and record the evaporator saturation temperature (T_{evaporator, sat}) from the low side gauge.
- 13. Step 13-Using the dry-bulb thermocouple already in place, measure and record the suction line temperature $(T_{\text{suction, db}})$.
- 14. Step 14-Using the dry-bulb thermocouple already in place, measure and record the condenser (entering) air dry-bulb temperature (T_{condenser, db}).

The above measurements shall be used to adjust refrigerant charge and airflow as described in following sections.

RDK2.5 Refrigerant Charge Calculations

The Superheat Charging Method is used only for non-TXV systems equipped with fixed metering devices. These include capillary tubes and piston-type metering devices. The following steps describe the calculations to determine if the system meets the required refrigerant charge using the measurements described in <u>S</u>section <u>RD</u>2.4. If a system fails, then remedial actions must be taken. If the refrigerant charge is changed and the airflow has been previously tested and shown to pass, then the airflow shall be re-tested. Be sure to complete Steps 1 and 2 of Section <u>RD</u>2.4 before re-testing the airflow. Both the airflow and charge must be re-tested until they both sequentially pass.

1. Step 1Calculate Actual Superheat as the suction line temperature minus the evaporator saturation temperature.

Actual Superheat = $T_{\text{suction, db}} - T_{\text{evaporator, sat}}$.

- 2. Step 2-Determine the Target Superheat using Table RD2K-1 using the return air wet-bulb temperature (T_{return}, wb) and condenser air dry-bulb temperature (T_{condenser}, db).
- 3. Step 3-If a dash mark is read from Table RD-2Table K-1, the target superheat is less than 5°F, then the system does not pass the required refrigerant charge criteria, usually because outdoor conditions are too hot and dry. One of the following adjustments is needed until a target superheat value can be obtained from Table RD-2Table K-1 by either 1) turning on the space heating system and/or opening the windows to warm up indoor temperature; or 2) retest at another time when conditions are different. After adjustments, repeat the measurement procedure as often as necessary to establish the target superheat. Allow system to stabilize for 15 minutes before completing the measurement procedure again.
- 4. Step 4-Calculate the difference between actual superheat and target superheat (Actual Superheat Target Superheat)
- 5. Step 5If the difference is between minus 5 and plus 5°F, then the system **passes** the required refrigerant charge criteria.
- 6. Step 6-If the difference is greater than plus 5°F, then the system **does not pass** the required refrigerant charge criteria and the installer shall add refrigerant. After the refrigerant has been added, turn the system on and allow it to stabilize for 15 minutes before completing the measurement procedure again. Adjust refrigerant charge and repeat the measurement procedure as many times as necessary to pass the test.
- 7. Step 7If the difference is between -5 and -100°F, then the system **does not pass** the required refrigerant charge criteria, the installer shall remove refrigerant. After the refrigerant has been removed, turn the system on and allow it to stabilize for 15 minutes before completing the measurement procedure again. Adjust refrigerant charge and repeat the measurement as many times as necessary to pass the test.

RDK2.65 Adequate Airflow Calculations Verification

In order to have a valid charge test, the air flow shall be verified by either passing the temperature split test or by one of the three measurements in ACM RE-2005 with a measured airflow in excess of 0.033 cfm/Btu capacity rated at DOE A test conditions (400 cfm/12000 Btu) (dry coil).

The temperature split <u>test</u> method is designed to provide an efficient check to see if airflow is above the required minimum <u>for a valid refrigerant charge test</u>. The following steps describe the calculations using the measurement procedure described in <u>S</u>ection <u>RD</u>2.4. If a system fails, then remedial actions must be taken. If the airflow is changed and the refrigerant charge has previously been tested and shown to pass, then the refrigerant charge shall be re-tested. Be sure to complete Steps 1 and 2 of Section <u>RD</u>2.4 before re-testing the refrigerant charge. Both the airflow and charge must be re-tested until they both sequentially pass.

- 1. Step 1Calculate the Actual Temperature Split as the return air dry-bulb temperature minus the supply air dry-bulb temperature. Actual Temperature Split = $T_{return, db}$ $T_{supply, db}$
- 2. Step 2Determine the Target Temperature Split from Table RD-3Table RK-2 using the return air wet-bulb temperature (T_{return, wb}) and return air dry-bulb temperature (T_{return, db}).
- 3. Step 3If a dash mark is read from Table RD-3Table RK-2, then there probably was an error in the measurements because the conditions in this part of the table would be extremely unusual. If this happens, re-measure the temperatures. If re-measurement results in a dash mark, complete one of the alternate airflow measurements in Section RE4.1 Section RD3.4 below.
- 4. Step 4Calculate the difference between target and actual temperature split (Actual Temperature Split-Target Temperature Split). If the difference is within plus 3°F and minus 3°F, then the system **passes** the adequate airflow criteria.
- 5. Step 5If the difference is greater than plus 3°F, then the system **does not pass** the adequate airflow criteria and the airflow shall be increased by the installer. Increasing airflow can be accomplished by eliminating restrictions in the duct system, increasing blower speed, cleaning filters, or opening registers. After corrective

measures are taken, repeat measurement procedure as often as necessary to establish adequate airflow range. Allow system to stabilize for 15 minutes before repeating measurement procedure.

- 6. Step-6If the difference is between minus 3°F and minus 100°F, then the measurement procedure shall be repeated making sure that temperatures are measured at the center of the airflow.
- 7. Step 7If the re-measured difference is between plus 3°F and minus 3°F the system **passes** the adequate airflow criteria. If the re-measured difference is between minus 3°F and minus 100°F, the system passes, but it is likely that_the capacity is low on this system (it is possible, but unlikely, that airflow is higher than average).

RDK3 Alternate Charge and Airflow Measurement Procedure

This section specifies the Alternate charge and airflow measurement procedure. Under this procedure, the required refrigerant charge is calculated using the *Weigh-In Charging Method*.

and adequate airflow across the evaporator coil is calculated using the Measured Airflow Method.

HVAC installers who must complete system installation verification when the outdoor temperature is below 55°F shall use this Alternate procedure in conjunction with installing and charging the system in accordance with the manufacturer's specifications. HERS Raters shall not use this procedure to verify compliance.

Split system air conditioners come from the factory already charged with the standard charge indicated on the name plate. The manufacturer supplies the charge proper for the application based on their standard liquid line length. It is the responsibility of the HVAC installer to ensure that the charge is correct for each air conditioner and to adjust the charge based on liquid line length different from the manufacturer's standard.

RDK3.1 Minimum Qualifications for this Procedure

HVAC installation technicians need toshall be qualified to perform the following:

- 1. Step 1Transfer and recovery of refrigerant (including a valid Environmental Protection Agency (EPA) certification for transition and recovery of refrigerant).
- 2. Step 2Accurately weigh the amount of refrigerant added or removed using an electronic scale.
- 3. Step 3Calculate the refrigerant charge adjustment needed to compensate for non-standard lineset lengths/diameters based on the actual lineset length/diameter and the manufacturer's specifications for adjusting refrigerant charge for non-standard lineset lengths/diameters.

RDK3.2 Instrumentation Specifications

Instrumentation for the procedures described in this section shall conform to the following specifications.

3.2.1 Digital Charging Scale

The digital scale used to weigh in refrigerant must have a range of .5 oz to at least 1200 oz (75 lb.). The scale's accuracy must be \pm 0.25 oz.

RDK3.3 Weigh-In Method

The following procedure shall be used by the HVAC installer to charge the system with the correct refrigerant charge.

- 1. Step 10btain manufacturer's standard liquid line length and charge adjustment for alternate liquid line lengths.
- 2. Step 2 Measure and record the actual liquid line length (L actual).
- 3. Step 3Record the manufacturer's standard liquid line length (L standard).
- 4. Step 4Calculate the difference between actual and standard liquid line lengths

- 5. Step 5Record the manufacturer's adjustment for liquid line length difference per foot (A length).
- 6. Step-6Calculate the amount of refrigerant to add or remove and document the calculations on the CF-6R.
- 7. Step 7Weigh in or remove the correct amount of refrigerant

3.4 Airflow Measurement

The airflow across the indoor evaporator coil shall be measured using one of the 2 methods described Appendix F - Standard Procedure for Determining the Seasonal Energy Efficiencies of Residential Air Distribution Systems:

Section 4.3.7.2.1 Diagnostic Fan Flow Using Flow Hood

Section 4.3.7.2.2 Diagnostic Fan Flow Using Plenum Pressure Matching

3.5 Adequate Airflow Calculation

The measured airflow method is used to provide a check to see if airflow is above the required minimum of 385 CFM per nominal ton of capacity (assumes coil is dry). The following steps describe the calculations using the measurement procedure described in Section 3.4. If a system fails, then remedial actions must be taken. The airflow must be re-tested until it passes.

Step 1. Record the measured airflow (F_measured) obtained from the measurement procedures described in Section 3.4.

Step 2. Obtain and record the rated cooling capacity (C cooling) in Btu.

Step 3. Calculate the required airflow as the product of the rated cooling capacity in Btu times 0.032.

Step 4. Compare the airflow measured according to section 3.4 with the required airflow.

Step 5. If the measured airflow is greater than the required airflow, then the system passes the adequate airflow criteria.

<u>Step 6.</u> If the measured airflow is less than the required airflow, the system does not pass the adequate airflow criteria and the airflow shall be increased by the installer. Increasing airflow can be accomplished by eliminating restrictions in the duct system, increasing blower speed, cleaning filters, or opening registers. After corrective measures are taken, repeat measurement procedure.

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Table <u>RD-K-2</u>4: Target Superheat (Suction Line Temperature - Evaporator Saturation Temperature)

Tubi	іе <u>кр</u>	- K <u>- L</u>	. rui	gor c	ирсп	rout	Cuor	1011 L	1110 1	ompe						<i>Ге</i> тр		•										
														(T	return,	wb)												
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	55	8.8	10.1	11.5	12.8	14.2	15.6	17.1	18.5	20.0	21.5	23.1	24.6	26.2	27.8	29.4	31.0	32.4	33.8	35.1	36.4	37.7	39.0	40.2	41.5	42.7	43.9	45.0
	56	8.6	9.9	11.2	12.6	14.0	15.4	16.8	18.2	19.7	21.2	22.7	24.2	25.7	27.3	28.9	30.5	31.8	33.2	34.6	35.9	37.2	38.5	39.7	41.0	42.2	43.4	44.6
	57	8.3	9.6	11.0	12.3	13.7	15.1	16.5	17.9	19.4	20.8	22.3	23.8		26.8			31.3	32.6	34.0	35.3	36.7		39.2	40.5	41.7	43.0	44.2
	58	7.9	9.3	10.6	12.0	13.4	14.8	16.2	17.6	19.0	20.4		23.3	24.8		27.8			32.1		34.8	36.1	37.5	38.7	40.0	41.3	42.5	43.7
	59	7.5	8.9	10.2	11.6	13.0	14.4	15.8	17.2	18.6	20.0	21.4		24.3	25.7		28.7		31.5		34.3		36.9	38.3	39.5	40.8	42.1	43.3
	60	7.0	8.4	9.8	11.2	12.6	14.0	15.4	16.8	18.2	19.6	21.0	22.4	23.8	25.2	26.6	28.1		31.0	32.4	33.7	35.1	36.4	37.8	39.1	40.4	41.6	
	61	6.5	7.9	9.3	10.7	12.1	13.5	14.9	16.3	17.7	19.1		21.9		24.7		27.5		30.4		33.2		35.9		38.6	39.9	41.2	
	62	6.0	7.4	8.8	10.2	11.7	13.1	14.5	15.9	17.3	18.7		21.4		24.2			28.4	29.9		32.7	34.1				39.4		42.0
	63	5.3	6.8	8.3	9.7	11.1	12.6	14.0	15.4	16.8	18.2		20.9				26.4		29.3		32.2		34.9		37.7	39.0	40.3	
	64	-	6.1	7.6	9.1	10.6		13.5	14.9	16.3	17.7					24.4			28.7		31.6		34.4		37.2	38.5	39.9	
(°F)	65	-	5.4	7.0	8.5	10.0	11.5	12.9	14.3	15.8	17.1	18.5	19.9	21.2	22.5	23.8	25.2		28.2	29.7	31.1	32.5	33.9	35.3	36.7	38.1	39.4	40.8
) e	66	-		6.3	7.8	9.3	10.8	12.3	13.8	15.2	16.6				22.0		24.6		27.6	29.1	30.6	32.0						40.4
tur	67	-	-	5.5	7.1	8.7	10.2	11.7	13.2	14.6	16.0		18.8		21.4		24.1		27.1		30.1	31.5		34.4		37.2		
era	68	-	-	-	6.3	8.0	9.5	11.1		14.0	15.5				20.8			25.0			29.5							39.5
ď	69	-	-	-	5.5	7.2	8.8	10.4	11.9	13.4	14.8				20.3 19.7	20.9	22.9	24.4 23.9	26.0 25.4	27.5 27.0	29.0 28.5	30.5		33.4				39.1 38.7
<u>F</u>	70	-	-	-	-	6.4	8.1	9.7	11.2	12.7	14.2	15.7	17.0						24.9			30.0 29.5		33.0 32.5	34.4	35.9 35.4		
Air Dry-Bulb Temperature	71 72	-	-	-	_	5.6	7.3	8.9	10.5	12.1	13.6	15.0	16.4		19.1		21.7			26.4	28.0			32.5	34.0	35.4		38.3 37.9
ģ	73	-	-	-	_	-	6.4	8.1 7.3	9.8 9.0	11.4 10.7	12.9 12.2	14.4 13.7	15.8 15.2		18.5 17.9	19.7 19.2	21.2 20.6		24.3 23.8		27.4 26.9	29.0 28.5		31.5	33.5 33.1	34.6	36.5 36.0	
ŗ	74	-	-	_	_	-	5.6	6.5	8.2	9.9	11.5	13.1			17.9			21.6					29.5			34.1		
Ë	75			_	_			5.6	7.4	9.2	10.8	12.4	13.9	15.3	16.7	18.0	19.4		22.7	24.3	25.9	27.5	29.1	30.6	32.2	33.7	35.2	36.7
¥ .	76			_				J.U	6.6	8.4	10.0	11.7	13.2	14.7	16.1	17.4	18.9		22.1	23.8	25.4	27.0		30.1		33.3		36.3
Condenser	77	_	_	_	_	_	_	_	5.7	7.5	9.3	11.0	12.5	14.0	15.4		18.3		21.6		24.9	26.5		29.7				36.0
de	78	_	_	_	_	_	_	_	-	6.7	8.5	10.2	11.8		14.8		17.7			22.7	24.4	26.0		29.2		32.4		
Š	79	_	_	_	_	_	_	_	_	5.9	7.7	9.5	11.1		14.2		17.1				23.8			28.8				
-	80	-	_	-	_	_	-	_	-	-	6.9	8.7	10.4	12.0	13.5		16.6		20.0		23.3	25.0		28.3	29.9		33.2	
	81	-	-	-	-	-	-	-	-	-	6.0	7.9	9.7	11.3	12.9	14.3	16.0		19.4	21.1	22.8	24.5	26.2	27.9	29.5	31.2	32.8	34.4
	82	-	-	-	_	-	-	-	-	-	5.2	7.1	8.9	10.6	12.2		15.4		18.9		22.3	24.0		27.4		30.7	32.4	34.0
	83	-	-	-	-	-	-	-	-	-	-	6.3	8.2	9.9	11.6	13.1	14.9	16.6	18.4	20.1	21.8	23.5	25.2	26.9	28.6	30.3	32.0	33.7
	84	-	-	-	-	-	-	-	-	-	-	5.5	7.4	9.2	10.9	12.5	14.3	16.1	17.8	19.6	21.3		24.8	26.5	28.2			33.3
	85	-	-	-		-	-	-	-	-	-	-	6.6	8.5	10.3		13.7		17.3	19.0		22.6		26.0				32.9
	86	-	-	-	-	-	-	-	-	-	-	-	5.8	7.8	9.6	11.3	13.2	15.0	16.7	18.5	20.3	22.1	23.8	25.6	27.3	29.1	30.8	32.6
	87	-	-	-	-	-	-	-	-	-	-	-	5.0	7.0	8.9	10.6	12.6	14.4	16.2	18.0	19.8	21.6	23.4	25.1	26.9	28.7	30.4	32.2
	88	-	-	-	-	-	-	-	-	-	-	-	-	6.3	8.2	10.0	12.0	13.9	15.7	17.5	19.3	21.1	22.9	24.7	26.5	28.3	30.1	31.8
	89	-	-	-	-	-	-	-	-	-	-	-	-	5.5	7.5	9.4	11.5	13.3	15.1	17.0	18.8	20.6	22.4	24.3	26.1	27.9	29.7	31.5
	90	-	-	-	-	-	-	-	-	-	-	-	-	-	6.8	8.8	10.9	12.8	14.6	16.5	18.3	20.1	22.0	23.8	25.6	27.5	29.3	31.1

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Greyed area indicates test conditions where the return drybulb temperature must exceed 70°F	
OTOYOU AFOU MAIOULO LOCE CONTAINION WHOTO THE TOTAL AT SAID LOTTIPOTALATO THACE OXOCOU TO T	

005 Residential ACM Manual	Page RD- 10
able <u>RD-K-2</u> 1: Target Superheat (Suction Line Temperature - Evaporator Saturation Temperature) (continued)	
opendix RD – Procedures for Determining Refrigerant Charge for Split System Space Cooling Systems without Thermost	

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												Retur	n Air	Wet-I	Bulb 1	Гетр	eratu	re (°F)									
													1	(T	return, v	_{vb})							1		1	1		
		50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 - - - - - - - - 6.1 8.1 10.3 12.2 14.1 15.9 17.8 19.7 21.5 23.4 25.2 27.1 28.9															76											
	91	-	-	-	-	-	-	-	-	-	-	-	-	-	6.1	8.1	10.3	12.2	14.1	15.9	17.8	19.7	21.5	23.4	25.2	27.1	28.9	30.8
	92	-	-	-	-	-	-	-	-	-	-	-	-	-	5.4	7.5	9.8	11.7	13.5	15.4	17.3	19.2	21.1	22.9	24.8	26.7	28.5	30.4
	93	-	-	-	-	-	-	-	-	-	-	-	-	-	-	6.8	9.2	11.1	13.0	14.9	16.8	18.7	20.6	22.5	24.4	26.3	28.2	30.1
	94	-	-	-	-	-	-	-	-	-	-	-	-	-	-	6.2	8.7	10.6	12.5	14.4	16.3	18.2	20.2	22.1	24.0	25.9	27.8	29.7
	95	•	-	-	-	-	-	-	•	-	-	-	-	-	-	5.6	8.1	10.0	12.0	13.9	15.8	17.8	19.7	21.6	23.6	25.5	27.4	29.4
	96	-	-	-	-	-	1		•	-	-	-	-	-	-	-	7.5	9.5	11.4	13.4	15.3	17.3	19.2	21.2	23.2	25.1	27.1	29.0
	97	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	7.0	8.9	10.9	12.9	14.9	16.8	18.8	20.8	22.7	24.7	26.7	28.7
F	98	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	6.4	8.4	10.4	12.4	14.4	16.4	18.3	20.3	22.3	24.3	26.3	28.3
lre (99	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	5.8	7.9	9.9	11.9	13.9	15.9	17.9	19.9	21.9	24.0	26.0	28.0
ratı	100	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	5.3	7.3	9.3	11.4	13.4	15.4	17.5	19.5	21.5	23.6	25.6	27.7
mpe	101	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	6.8	8.8	10.9	12.9	15.0	17.0	19.1	21.1	23.2	25.3	27.3
b Te	102	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	6.2	8.3	10.4	12.4	14.5	16.6	18.6	20.7	22.8	24.9	27.0
Bull	103	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	5.7	7.8	9.9	11.9	14.0	16.1	18.2	20.3	22.4	24.5	26.7
Dry	104	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	5.2	7.2	9.3	11.5	13.6	15.7	17.8	19.9	22.1	24.2	26.3
Condenser Air Dry-Bulb Temperature (°F)	105	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	6.7	8.8	11.0	13.1	15.2	17.4	19.5	21.7	23.8	26.0
nser	106	-	-	-	-	-	-		-	-	-	-	-	-	-	-	-	-	6.2	8.3	10.5	12.6	14.8	17.0	19.1	21.3	23.5	25.7
nde	107	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	5.7	7.9	10.0	12.2	14.4	16.6	18.7	21.0	23.2	25.4
ပိ	108	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	5.2	7.4	9.5	11.7	13.9	16.1	18.4	20.6	22.8	25.1
	109	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	6.9	9.1	11.3	13.5	15.7	18.0	20.2	22.5	24.7
	110	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	6.4	8.6	10.8	13.1	15.3	17.6	19.9	22.1	24.4
	111	-	-	-	-	-	-	1	-	-	-	-	-	-	-	-	-	-	-	5.9	8.1	10.4	12.6	14.9	17.2	19.5	21.8	24.1
	112	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	-	5.4	7.6	9.9	12.2	14.5	16.8	19.1	21.5	23.8
	113	-	_	-	-	-	-		-	-	-	-	-	-	-	•	-	-	-	-	7.2	9.5	11.8	14.1	16.4	18.8	21.1	23.5
	114	-	-	-	-	-	-		-	-	-	-	-	-	-	•	-	-	-	•	6.7	9.0	11.4	13.7	16.1	18.4	20.8	23.2
	115	-	-	-	-	-	-		-	-	-	-	-	-	-	-	-	-	-	-	6.2	8.6	10.9	13.3	15.7	18.1	20.5	22.9

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Table RD-K-32: Target Temperature Split (Return Dry-Bulb – Supply Dry-Bulb)

	Return Air Wet-Bulb (°F) (T return, wb)																											
		50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76
	70	20.9	20.7	20.6	20.4	20.1	19.9	19.5	19.1	18.7	18.2	17.7	17.2	16.5	15.9	15.2	14.4	13.7	12.8	11.9	11.0	10.0	9.0	7.9	6.8	5.7	4.5	3.2
	71	21.4	21.3	21.1	20.9	20.7	20.4	20.1	19.7	19.3	18.8	18.3	17.7	17.1	16.4	15.7	15.0	14.2	13.4	12.5	11.5	10.6	9.5	8.5	7.4	6.2	5.0	3.8
	72	21.9	21.8	21.7	21.5	21.2	20.9	20.6	20.2	19.8	19.3	18.8	18.2	17.6	17.0	16.3	15.5	14.7	13.9	13.0	12.1	11.1	10.1	9.0	7.9	6.8	5.6	4.3
, db)	73	22.5	22.4	22.2	22.0	21.8	21.5	21.2	20.8	20.3	19.9	19.4	18.8	18.2	17.5	16.8	16.1	15.3	14.4	13.6	12.6	11.7	10.6	9.6	8.5	7.3	6.1	4.8
return, db)	74	23.0	22.9	22.8	22.6	22.3	22.0	21.7	21.3	20.9	20.4	19.9	19.3	18.7	18.1	17.4	16.6	15.8	15.0	14.1	13.2	12.2	11.2	10.1	9.0	7.8	6.6	5.4
F)	75	23.6	23.5	23.3	23.1	22.9	22.6	22.2	21.9	21.4	21.0	20.4	19.9	19.3	18.6	17.9	17.2	16.4	15.5	14.7	13.7	12.7	11.7	10.7	9.5	8.4	7.2	5.9
Dry-Bulb (°F)	76	24.1	24.0	23.9	23.7	23.4	23.1	22.8	22.4	22.0	21.5	21.0	20.4	19.8	19.2	18.5	17.7	16.9	16.1	15.2	14.3	13.3	12.3	11.2	10.1	8.9	7.7	6.5
Bull	77	-	24.6	24.4	24.2	24.0	23.7	23.3	22.9	22.5	22.0	21.5	21.0	20.4	19.7	19.0	18.3	17.5	16.6	15.7	14.8	13.8	12.8	11.7	10.6	9.5	8.3	7.0
J.Y	78	-	-	-	24.7	24.5	24.2	23.9	23.5	23.1	22.6	22.1	21.5	20.9	20.2	19.5	18.8	18.0	17.2	16.3	15.4	14.4	13.4	12.3	11.2	10.0	8.8	7.6
Air [79	-	-	-	-	-	24.8	24.4	24.0	23.6	23.1	22.6	22.1	21.4	20.8	20.1	19.3	18.5	17.7	16.8	15.9	14.9	13.9	12.8	11.7	10.6	9.4	8.1
Return	80	-	-	-	-	-	-	25.0	24.6	24.2	23.7	23.2	22.6	22.0	21.3	20.6	19.9	19.1	18.3	17.4	16.4	15.5	14.4	13.4	12.3	11.1	9.9	8.7
Ret	81	-	-	-		-	-	-	25.1	24.7	24.2	23.7	23.1	22.5	21.9	21.2	20.4	19.6	18.8	17.9	17.0	16.0	15.0	13.9	12.8	11.7	10.4	9.2
	82	-	-	-	-	-	-	-	-	25.2	24.8	24.2	23.7	23.1	22.4	21.7	21.0	20.2	19.3	18.5	17.5	16.6	15.5	14.5	13.4	12.2	11.0	9.7
	83	-	-	-	-	-	-	-	-	-	25.3	24.8	24.2	23.6	23.0	22.3	21.5	20.7	19.9	19.0	18.1	17.1	16.1	15.0	13.9	12.7	11.5	10.3
	84	-	-	-	-	-	-	-	-	-	25.9	25.3	24.8	24.2	23.5	22.8	22.1	21.3	20.4	19.5	18.6	17.6	16.6	15.6	14.4	13.3	12.1	10.8

APPENDIX D

The Contents of Appendix D Have Been Deleted.

Appendix D is Reserved for Future Use for Sample CALRES Test Run Files and Input Descriptions for the Optional Capabilities Tests

These sample files will be added for information purposes only, and will not be adopted as regulations.